

Methods for model error analysis of pulse impulse measuring torsion system

Xing Jin ^a, Dapeng Wang ^{b*} and Chao Zhu ^c

State Key Laboratory of Laser Propulsion & Application, Equipment Academy, Beijing 101416, China

^ajinxing_beijing@sina.com, ^bajwzajwz@163.com, ^cZhuchaod1630@163.com

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Abstract. Torsion system is a typical pulse impulse measuring method. Currently used analysis model is the impulse moment interaction model. According to the response characteristic analysis of torsion system under pulsed impulse action, using analytical methods, this thesis proposes error analysis methods for impulse momentary action model and also proposes the main factors of model error is damping ratio of torsion system, the ratio of pulse time as well as system impulse period. The proposed model error analysis methods facilitate pulse impulse error analysis, which provides analysis and design methods for impulse pulse measurement and error analysis in engineering.

Introduction

In engineering, the thrust is considered as impulse force when action time is short, then we measure impulse that thrust generated. For example, when action time is less than 1/4 of torsion vibration period, changes of thrust over time is difficult to identify due to the limited number of sampling points of system response. Then effects of thrust perform as impulse effect and thrust measurement is converted into impulse measurement ^[1, 2].

The current used model is momentary impulse action model, but the actual thrust always has action time, so there is a certain discrepancy between impulse measurement results and the actual situation, and it is needed to determine methods of analyzing model errors and the process of how model errors act so as to increase impulse measurement accuracy ^[3, 4].

Momentary impulse role model

Under moment impulse action, the torsion system response is shown below.

$$q(t) = \frac{SL_f}{Jw_d} e^{-zw_n t} \sin w_d t \quad (1)$$

where, L_f is the arm; z is damping ratio; w_d is the torsion vibration frequency ($w_d = \sqrt{1-z^2} w_n$); J is the moment of inertia.

The relationship of torsion angle $q(t_{Mk})$ and time t_{Mk} of system response curve's extreme points and impulse measured is shown in Eq. 2.

$$S_k = \frac{Jw_d e^{zw_n t_{Mk}}}{L_f \sin w_d t_{Mk}} q(t_{Mk}) \quad (k=1, 2, \dots, L) \quad (2)$$

If the j -th impulse measurement is S_j , the sample mean and sample standard deviation of the impulse is shown in Eq. 3.

$$\bar{S} = \frac{1}{n} \sum_{j=1}^n S_j, S_S = \sqrt{\frac{1}{n-1} \sum_{j=1}^n (S_j - \bar{S})^2} \quad (3)$$

where, n is the number of measurements, and it can be used to estimate the confidence interval. In the above, the thrust time is negligible and the system response is the effect of the momentary impulse, so impulse measurement and calculation methods are called impulse momentary action models^[5].

Error analysis methods of momentary impulse action model

Momentary impulse action model is an ideal model that regards the action time as infinitesimal (negligible), while the actual thrust always has action time. The below will research on errors of the momentary impulse action model.

Set the thrust as $f(t) = f_0 (0 \leq t \leq T_0)$, the action time of thrust as T_0 , the arm as L_f , the moment as $M(t) = f_0 L_f$, then the torsion system response is shown as below.

$$\begin{aligned} q_1(t) &= \frac{L_f}{Jw_d} \int_0^t f(t) e^{-zw_n(t-t)} \sin w_d(t-t) dt \\ &= \frac{f_0 T_0 L_f}{Jw_d T_0} \int_0^t e^{-zw_n(t-t)} \sin w_d(t-t) dt \\ &= \frac{SL_f}{Jw_d} \cdot \frac{1}{T_0} \int_0^t e^{-zw_n(t-t)} \sin w_d(t-t) dt \end{aligned} \quad (4)$$

Where, $S = f_0 T_0$ is relevant impulse of the thrust. When $t > T_0$, set initial torsion angle as $q_1(T_0)$, the initial angular velocity as $\dot{q}_1(T_0)$, then the system begin to vibrate freely, and the below is get.

$$\begin{aligned} q_1(T_0) &= \frac{SL_f}{Jw_d} \cdot \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \sin w_d(T_0-t) dt \\ \dot{q}_1(T_0) &= \frac{SL_f w_n}{Jw_d} \cdot \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \cos[w_d(T_0-t) + a_1] dt \\ a_1 &= \arctan \frac{zw_n}{w_d} = \arctan \frac{z}{\sqrt{1-z^2}} \end{aligned}$$

When $t > T_0$, the system response of free vibration stage is shown in Eq.5 below.

$$\begin{aligned} q_2(t-T_0) &= \sqrt{q_1^2(T_0) + \left[\frac{q_1(T_0)zw_n + \dot{q}_1(T_0)}{w_d} \right]^2} e^{-zw_n(t-T_0)} \sin[w_d(t-T_0) + a_2] \\ a_2 &= \arctan \frac{q_1(T_0)w_d}{q_1(T_0)zw_n + \dot{q}_1(T_0)} \end{aligned} \quad (5)$$

For ease of discussion, dimensionless quantity is introduced.

$$\begin{aligned} A &= \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \sin w_d(T_0-t) dt \\ B &= \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \cos[w_d(T_0-t) + a_1] dt \end{aligned}$$

Then the below can be get.

$$\begin{aligned} q_1(T_0) &= \frac{SL_f}{Jw_d} \cdot \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \sin w_d(T_0-t) dt = \frac{SL_f A}{Jw_d} \\ \frac{q_1(T_0)zw_n}{w_d} &= \frac{SL_f zw_n}{Jw_d^2} \cdot \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \sin w_d(T_0-t) dt = \frac{SL_f zw_n A}{Jw_d^2} \\ \frac{\dot{q}_1(T_0)}{w_d} &= \frac{SL_f w_n}{Jw_d^2} \cdot \frac{1}{T_0} \int_0^{T_0} e^{-zw_n(T_0-t)} \cos[w_d(T_0-t) + a_1] dt = \frac{SL_f w_n B}{Jw_d^2} \end{aligned}$$

$$\sqrt{q_1^2(T_0) + \left[\frac{q_1(T_0)zw_n + q_1(T_0)}{w_d} \right]^2} = \frac{SL_f}{Jw_d} \sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2}$$

$$a_2 = \arctan \frac{q_1(T_0)w_d}{q_1(T_0)zw_n + q_1(T_0)} = \arctan \frac{\sqrt{1-z^2}A}{zA+B}$$

Thus, the free vibration Equation is rewritten as below.

$$q_2(t-T_0) = \frac{SL_f}{Jw_d} \sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{-zw_n(t-T_0)} \sin[w_d(t-T_0) + a_2]$$

As described above, on conditions of impulse S and action time T_0 , the system response Equation is different from momentary impulse action model.

The time and torsion angles corresponding to the extreme points of system response curves are given respectively as below.

$$w_d(t_{Mk} - T_0) + a_2 = \frac{(2k-1)\pi}{2} \quad (k=1,2,\mathbf{L})$$

$$q_2(t_{Mk} - T_0) = \frac{SL_f}{Jw_d} \sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{-\frac{z}{\sqrt{1-z^2}} \left[\frac{(2k-1)\pi}{2} - a_2 \right]} \quad (k=1,2,\mathbf{L})$$

Take t_{Mk} and $q_2(t_{Mk} - T_0)$ (equivalent to actual measurement results) into momentary impulse action model , then the value of estimated impulse S' can be expressed as below.

$$\frac{SL_f}{Jw_d} \sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{-\frac{z}{\sqrt{1-z^2}} \left[\frac{(2k-1)\pi}{2} - a_2 \right]} = \frac{S'L_f}{Jw_d} e^{-zw_n t_{Mk}} |\sin w_d t_{Mk}|$$

The actual acted impulse is S . According to momentary impulse action model, the estimated impulse is S' , and then the relative error of measuring impulse is given as below.

$$e_s = \frac{S' - S}{S} = \frac{\sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{-\frac{z}{\sqrt{1-z^2}} \left[\frac{(2k-1)\pi}{2} - a_2 \right]} - 1}{e^{-\frac{z}{\sqrt{1-z^2}} \left[w_d T_0 + \frac{(2k-1)\pi}{2} - a_2 \right]} \left| \sin \left[w_d T_0 + \frac{(2k-1)\pi}{2} - a_2 \right] \right|} - 1$$

By combination of the above analysis, the relative error formula of momentary impulse action model is given as Eq. 6.

$$e_s = \frac{S' - S}{S} = \frac{\sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{\frac{z}{\sqrt{1-z^2}} w_d T_0}}{\left| \sin \left[w_d T_0 + \frac{(2k-1)\pi}{2} - a_2 \right] \right|} - 1 \quad (6)$$

where , $k=1,2,\mathbf{L}$ respectively corresponds to the first maximum value, the first minimum value, the second maximum value and the second minimum value, etc. , so is the after.

The action time of is expressed as multiple of system cycles $T = 2\pi / w_d$: $T_0 = k_T T = k_T (2\pi / w_d)$, then the calculation formula of model errors is given as below.

$$e_s = \frac{S' - S}{S} = \frac{\sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{\frac{z}{\sqrt{1-z^2}} (2\pi k_T)}}{\left| \sin \left[2\pi k_T + \frac{(2k-1)\pi}{2} - a_2 \right] \right|} - 1 = \frac{\sqrt{A^2 + \left(\frac{zA+B}{\sqrt{1-z^2}} \right)^2} e^{\frac{z}{\sqrt{1-z^2}} (2\pi k_T)}}{\left| \sin \left[2\pi k_T - \frac{\pi}{2} - a_2 \right] \right|} - 1 \quad (7)$$

Obviously, the main error factors of momentary impulse action model is damping ratio and cycle multiples k_T , and independent of other factors.

Analysis and discussion

Fig. 1 and Fig. 2 shows relative errors of momentary impulse action model changes with variation of torsion damping ratio and cycle multiples.

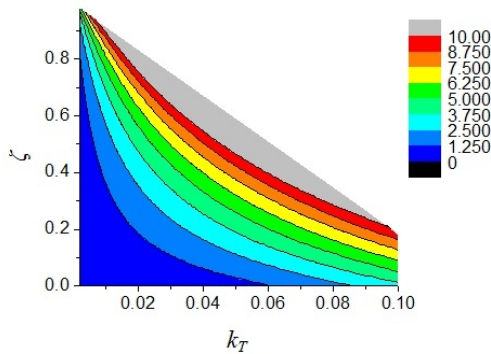


Fig. 1 relative errors(%)changes with k_T and ζ

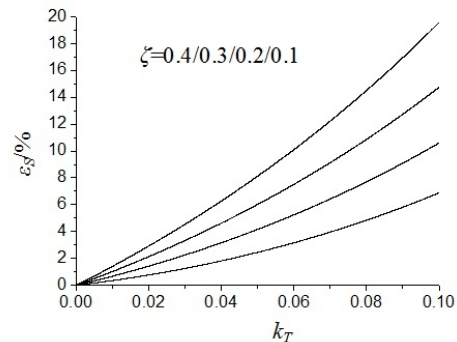


Fig. 2 relative errors changes with k_T

Figure 1 shows the relative error e_s (expressed in %) increases with increasing of k_T and ζ . The calculation results show when $k_T = 0.1$, the relative error range is $6.9\% \leq e_s \leq 19.6\%$ within usual damping ratio range $0.1 \leq \zeta \leq 0.4$.

Figure 2 shows on conditions of $\zeta = 0.4/0.3/0.2/0.1$, the relative error e_s increases with increasing of k_T .

Summary

Based on theoretically modeling, the thesis analyzes and discusses the model errors of momentary impulse action model, which is commonly used in engineering. Specific conclusions are as follows:

- (1) The measuring principle of momentary impulse action model is based on that the thrust action time is ignored, in specific, less than 1/4 of the pendulum cycle, and the smaller the action time is, and the more accurate are measurements.
- (2) The actual thrust always has a certain action time, which deviates from momentary impulse action model, and such deviation is exactly the momentary impulse action model errors. The smaller is the ratio of thrust action time and system cycle, the smaller are the model errors.
- (3) Main applications of impulse measurement are: 1) concerning only impulse value, ignoring impulse variation over time, for example, controlling impulse according to the impulse bits; 2) hard to identify impulse variation over time due to too small ratio of thrust action time and system cycle.

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