

3 types of vehicle-bridge coupling vibration analysis of model comparison

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Abstract: Compares two 1/4 car models, two 1/4 model (changes the spring stiffness) and 1/2 car models three vehicles at the same speed in the displacement effect. Comparison: when the vehicle speed is less than 130km/h, and displacement response curves in the three models are calculated when a highway bridge coupling vibration analysis of vehicle is two one-fourth model can be used to simulate.

Introduction

Vehicle-bridge coupled vibration attracts many researchers' attention as early as 100 years ago. The current study has simplified the vehicle and bridge model in it to make for a simpler solution.

In recent years, many Chinese researchers are trying to use numerical method to solve coupling problems. The document^[4] builds overall 3D model to make the interaction between vehicle and bridge as a whole, thus establishing differential equations, with the help of numerical methods, the dynamic response is obtained with respect to simple-supported girder and continuous beam in highway. The document^[5-6] makes a model of vehicle bridge coupling of simple-supported beam bridge employing d'alembert's principle and uses numerical methods to solve the problem. Step-by-step integral is used to calculate the coupling system consisting of mass with uniform variable speed and a simple supported beam. The document^[8] acquires the power series solution to Willis equation. The document^[9] derives the vibration equation by ignoring moving constant loads of mass vehicle and thus gets the exact solution. The document^[10] uses concentrated mass as the simplified bridge model and adds 2D multi-axis trailer load.

To investigate the influence of vehicle model on vehicle-bridge coupling, three kinds of models: two quarter-car models (spring stiffness changed) and half-car models with different velocity, are used to analyze displacement response. The overall laws are analyzed in the first place which followed the analysis of difference of models.

Model

one-fourth car models

Figure 1 is vehicle model, the spring-damper-mass system is for simulating vehicles. Where m_b is vehicle mass. m_t is the frame and wheel mass; k_a is vertical stiffness; c_a is vertical damping; k_b is vertical stiffness; c_b is vertical damping; v is the speed when vehicles go cross the bridge.

The vibration equation of vehicle and bridge are as follows:

$$m_t \frac{d^2 y_t}{dt^2} + m_b \frac{d^2 y_b}{dt^2} + c_a \left(\frac{dy_t}{dt} + \frac{dy_w}{dt} \right) + k_a (y_t + w) = 0,$$

$$m_b \frac{d^2 y_b}{dt^2} + c_b \left(\frac{dy_b}{dt} + \frac{dy_t}{dt} \right) + k_b (y_b - y_t) = 0.$$

Supposing $w(x,t) = \sum_n X_n(x)T_n(t)$ by using variable separation method. Using simple

supported beam as boundary condition and presuming that $X_n(x) = \sin(np\pi x/l)$. According the orthogonality of principal mode, the vibration differential equation under loads is as follows:

$$\ddot{T}_n(t) + \omega_n^2 T_n = \frac{2(m_t + m_b)g}{ml} \sin \frac{pvt}{l} +$$

$$\ddot{T}_n(t) + \frac{2m_t}{ml} \frac{dy_t}{dt^2} \sin \frac{npvt}{l} + \frac{2m_b}{ml} \frac{d^2 y_b}{dt^2} \sin \frac{npvt}{l}$$

With the help of variable separation method, suppose $w(x,t) = \sum_n X_n(x)T_n(t)$. By

simply-supported beams with boundary condition, let $X_n(x) = \sin(np\pi x/l)$, Again according to the principal mode of orthogonality, get the vibration differential equation of the bridge under the action of load as follows:

$$\ddot{T}_n(t) + \omega_n^2 T_n = \frac{2(m_t + m_b)g}{ml} \sin \frac{pvt}{l} + \frac{2m_t}{ml} \frac{dy_t}{dt^2} \sin \frac{npvt}{l} + \frac{2m_b}{ml} \frac{d^2 y_b}{dt^2} \sin \frac{npvt}{l},$$

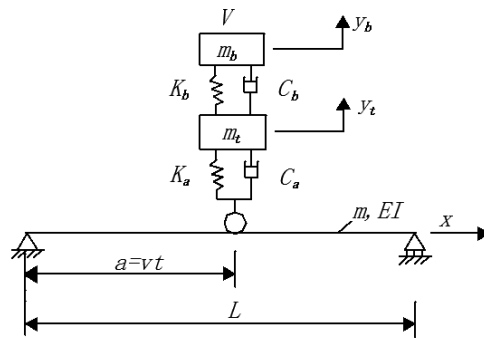


Fig. 1. One-fourth model under the effect of simply supported beam

Two quarter-car models will be used. The vibration equations for vehicle and bridge are taken into consideration separately, the results are in superposition. The other one is the same.

Half-car models

Figure 2 is vehicle model, the spring-damper-mass system is for simulation of vehicles. Where m_b is vehicle mass. I_b is the rigidity of vehicle body; m_t is the mass of wheels and structures; k_a is vertical stiffness; c_a is vertical damping; k_b is vertical stiffness; c_b is vertical damping; v is the speed when vehicles go cross the bridge (constant value). The vibration equations are as follows^[6]:

$$\frac{dT_n(t)}{dt^2} + w_n^2 T_n(t) = \frac{2p_1(t)}{ml} \sin \frac{np(vt-a)}{l} d_1(t) + \frac{2p_2(t)}{ml} \sin \frac{npvt}{l} d_2(t)$$

Which:

$$d_1(t) = \begin{cases} 1, & \frac{a}{v} \leq t \leq \frac{l+a}{v}; \\ 0, & \text{else} \end{cases}, \quad d_2(t) = \begin{cases} 1, & 0 \leq t \leq \frac{l}{v}; \\ 0, & \text{else} \end{cases}$$

Analysis of load of the vehicle system:

$$\left. \begin{aligned} p_1(t) &= m_{a1}g + \frac{m_b}{2}g + m_{a1}\ddot{y}_{t1} + \frac{m_s}{2}\ddot{y}_b - \frac{I_b}{a}\ddot{q} \\ p_2(t) &= m_{t2}g + \frac{m_b}{2}g + m_{t2}\ddot{y}_{t2} + \frac{m_b}{2}\ddot{y}_b - \frac{I_b}{a}\ddot{q} \end{aligned} \right\}$$

Analysis of load of the bogie:

$$m_{t1}\ddot{y}_{t1} + c_{a1}(\dot{y}_{t1} + \dot{w}_1) + k_{a1}(y_{t1} + w_1) + c_{b1}(\dot{y}_{t1} - \dot{y}_b + \frac{a}{2}\dot{q}) + k_{b1}(y_{t1} - y_b + \frac{a}{2}q) = 0$$

$$m_{t2}\ddot{y}_{t2} + c_{a2}(\dot{y}_{t2} + \dot{w}_2) + k_{a2}(y_{t2} + w_2) + c_{b2}(\dot{y}_{t2} - \dot{y}_b - \frac{a}{2}\dot{q}) + k_{b2}(y_{t2} - y_b - \frac{a}{2}q) = 0 \quad \text{Which:}$$

$$w_1 = \sum_n X_n(vt-a)T_n(t); \quad w_2 = \sum_n X_n(vt)T_n(t)$$

Analysis of load of the body:

$$m_b\ddot{y}_b + c_{b1}(\dot{y}_b - \dot{y}_{t1} - \frac{a}{2}\dot{q}) + k_{b1}(y_b - y_{t1} - \frac{a}{2}q) - c_{b2}(\dot{y}_{t2} - \dot{y}_b - \frac{a}{2}\dot{q}) - k_{b2}(y_{t2} - y_b - \frac{a}{2}q) = 0$$

$$I_b\ddot{q} - \frac{a}{2} \left[c_{b1}(\dot{y}_b - \dot{y}_{t1} - \frac{a}{2}\dot{q}) + k_{b1}(y_b - y_{t1} - \frac{a}{2}q) \right] + \frac{a}{2} \left[c_{b2}(\dot{y}_{t2} - \dot{y}_b - \frac{a}{2}\dot{q}) + k_{b2}(y_{t2} - y_b - \frac{a}{2}q) \right] = 0.$$

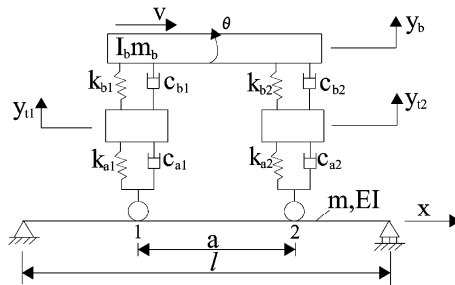


Fig. 2. Half-car model under the effect of simply supported beam

Results

Bridge data in document^[11] and vehicle data in document^[12] are used, the length of simple supported beam is $L=32m$. Mass per unit length $m=5.41 \times 10^3 kg/m$. flexural rigidity $EI=3.5 \times 10^{10} N \cdot m^2$. Following are relevant data of quarter-car model: $m_{t1}=m_{t2}=4330kg$; $m_b=19250kg$; $a=8.4m$; $k_{a1}=k_{a2}=4.28 \times 10^6 N/m$; $k_{b1}=k_{b2}=2.535 \times 10^6 N/m$; $c_{a1}=c_{a2}=9.8 \times 10^4 kg/s$; $c_{b1}=c_{b2}=1.96 \times 10^5 kg/s$. 1/2 car models: $c_{a1}=c_{a2}=9.8 \times 10^4 kg/s$; $m_b=38500kg$; $m_{t1}=m_{t2}=4330kg$; $I_b=2.446 \times 10^6 kg \cdot m^2$; $a=8.4m$;

$$k_{a1} = k_{a2} = 4.28 \times 10^6 \text{ N/m} ; k_{b1} = k_{b2} = 2.535 \times 10^6 \text{ N/m} ; c_{b1} = c_{b2} = 1.96 \times 10^5 \text{ kg/s} .$$

Figure 3~Figure 6 compare 3 kinds of vehicles vertical displacement at mid-point of beam at same speed. Figure 7 shows the largest displacement of the 3 kinds of models on the bridge under different speed. The comparison of curves in the figure shows that the 3 kinds of models could embody coupling response.

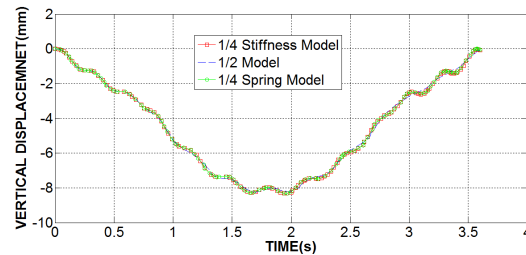


Fig. 3. Vertical displacement at mid-point of beam at speed 40km/h

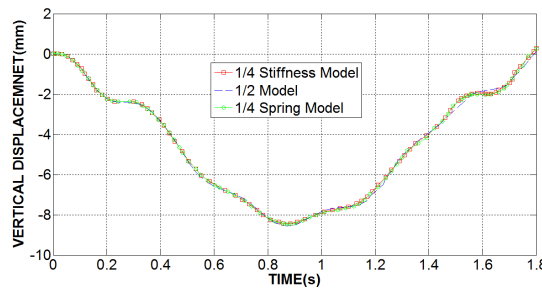


Fig. 4. Vertical displacement at mid-point of beam at speed 80km/h

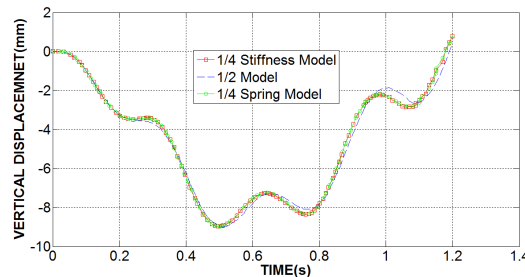


Fig. 5. Vertical displacement at mid-point of beam at speed 120km/h

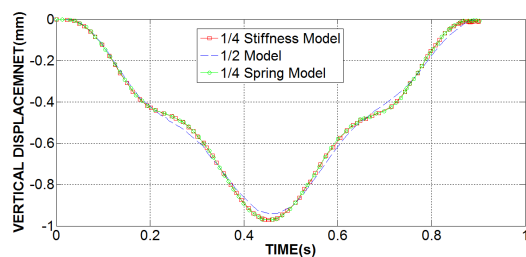


Fig. 6. Vertical displacement at mid-point of beam at speed 160km/h

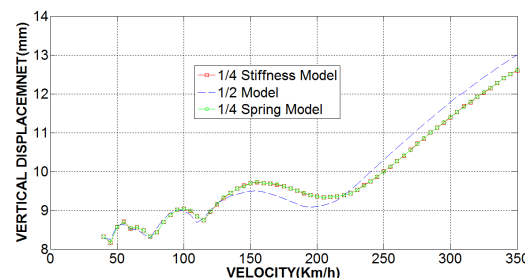


Fig. 7. The largest displacement on the bridge under different speed

General laws could be arrived at based on above figures:

- (1) They all give good expression to the vibration response.
- (2) Higher speed will make a slower displacement response of mid-span.
- (3) Response Maximum displacement in bridge when vehicle appear in mid-span.
- (4) When speed is under 200km/h, response of maximum displacement bridge do not take on a linear relationship.

The difference between vehicle models are as follows:

- (1) The difference between 1/4 spring model of rigidity and 1/4 spring model is the change of rigidity. The displacement response in mid-span of bridge fits well with each other. Spring stiffness has no impact on vehicle-bridge coupling.
- (2) When speed is under 130km/h, the discrepancy of maximum displacement in 1/4 vehicle model and 1/2 vehicle model is small.
- (3) When speed is under 220km/h, the maximum displacement in 1/2 vehicle model is smaller than that 1/2 vehicle model. The vehicle-bridge coupling is more sensitive to moment of inertia.
- (4) When speed is more than 220km/h, the maximum displacement in 1/2 vehicle model is greater than that 1/2 vehicle model.
- (5) Generally speaking, the vehicle speed is no more than 120/h, The vehicle-bridge coupling can be realized using 1/4 model in highway bridge.

Conclusions

To study the response of factor of vehicle model to the vehicle-bridge coupling, the displacement of mid-span of 3 kinds of models are calculated. Comparison shows that the 3 kinds of models are in good agreement with vibration response of vehicle-bridge coupling. With the add of rotational inertia, the agreement will be better yet make the analysis complicated. When velocity is under 130km/h, the 3 kinds of models are in good agreement with each other. Quarter-car models can be used to analyse vibration of vehicle-bridge coupling in highway bridge.

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